

Soit P la puissance de moteur

$$P = M_t \cdot \omega = 307.35 \text{ W}$$

$$N = 100 \text{ tr/ mn}$$

$$r = 45 \text{ mm}$$

$$IC = a \approx 1100 \text{ mm}$$

$$CD = b = 1000 \text{ mm}$$

Acier C35 (Re = 430 N/mm²)

$$\alpha = 20^\circ$$

$$\beta = 45^\circ$$

$F_t \tan \alpha$: Projection de la force de pression sur le plan (\vec{X}, \vec{Y})

On a :

$$\{\tau\}_I = \begin{cases} \vec{R} \\ \vec{M} \end{cases} ; \rightarrow_R \begin{cases} Fa = -F_t \cdot \tan \alpha \cos \beta \\ Fr = F_t \cdot \tan \alpha \sin \beta \\ F_t \end{cases} ; \vec{M} \begin{cases} 0 \\ 0 \\ 0 \end{cases}$$

$$P = C \cdot \omega = C \cdot \frac{N2\pi}{60} = F_t \cdot r \cdot \frac{N2\pi}{60}$$

$$F_t = \frac{60P}{2\pi N r}$$

$$\text{D'où } \{\tau\}_I = \begin{cases} \vec{R} \\ \vec{M} \end{cases} \text{ avec } \vec{R} \begin{cases} Fa = -F_t \cdot \tan \alpha \cos \beta \\ Fr = F_t \cdot \tan \alpha \sin \beta \\ F_t = \frac{60P}{2\pi N r} \end{cases}$$

$$\begin{cases} \sum \vec{F} = 0 \\ \sum \vec{M} = 0 \end{cases} \Rightarrow \vec{R}_I + \vec{R}_C + \vec{R}_D = \vec{0} \quad (1)$$

$$\vec{R}_I \wedge \vec{I}_c + \vec{R}_D \wedge \vec{DC} - \vec{C}_t = \vec{0} \quad (2)$$

$$(1) \begin{cases} F_a \\ F_r \\ F_t \end{cases} + \begin{cases} X_c \\ Y_c \\ Z_c \end{cases} + \begin{cases} 0 \\ Y_D \\ Z_D \end{cases} = \begin{cases} 0 \\ 0 \\ 0 \end{cases}$$

$$(2) \begin{cases} 0 \\ y_D \\ z_D \end{cases} \wedge \begin{cases} -b \\ 0 \\ 0 \end{cases} + \begin{cases} F_a \\ F_r \\ F_t \end{cases} \wedge \begin{cases} 0 \\ r \\ 0 \end{cases} - \begin{cases} C_t \\ 0 \\ 0 \end{cases} = \begin{cases} 0 \\ 0 \\ 0 \end{cases}$$

$$\begin{cases} Z_D = \frac{F_t \cdot a}{b} = 717,2N \\ Y_D = -\frac{F_b}{b} = -0,168N \\ X_C = -F_a = -0,168N \\ Y_C = -F_r + \frac{r}{b} \cdot F_a = -160,4N \\ Z_C = -F_t - \frac{a}{b} \cdot F_t = -1369,2N \end{cases} \quad \begin{cases} F_a = 0,168kN \\ F_r = 0,168kN \\ F_t = 0,652kN \end{cases}$$

Efforts intérieurs

On néglige les efforts normal et tranchant N et T

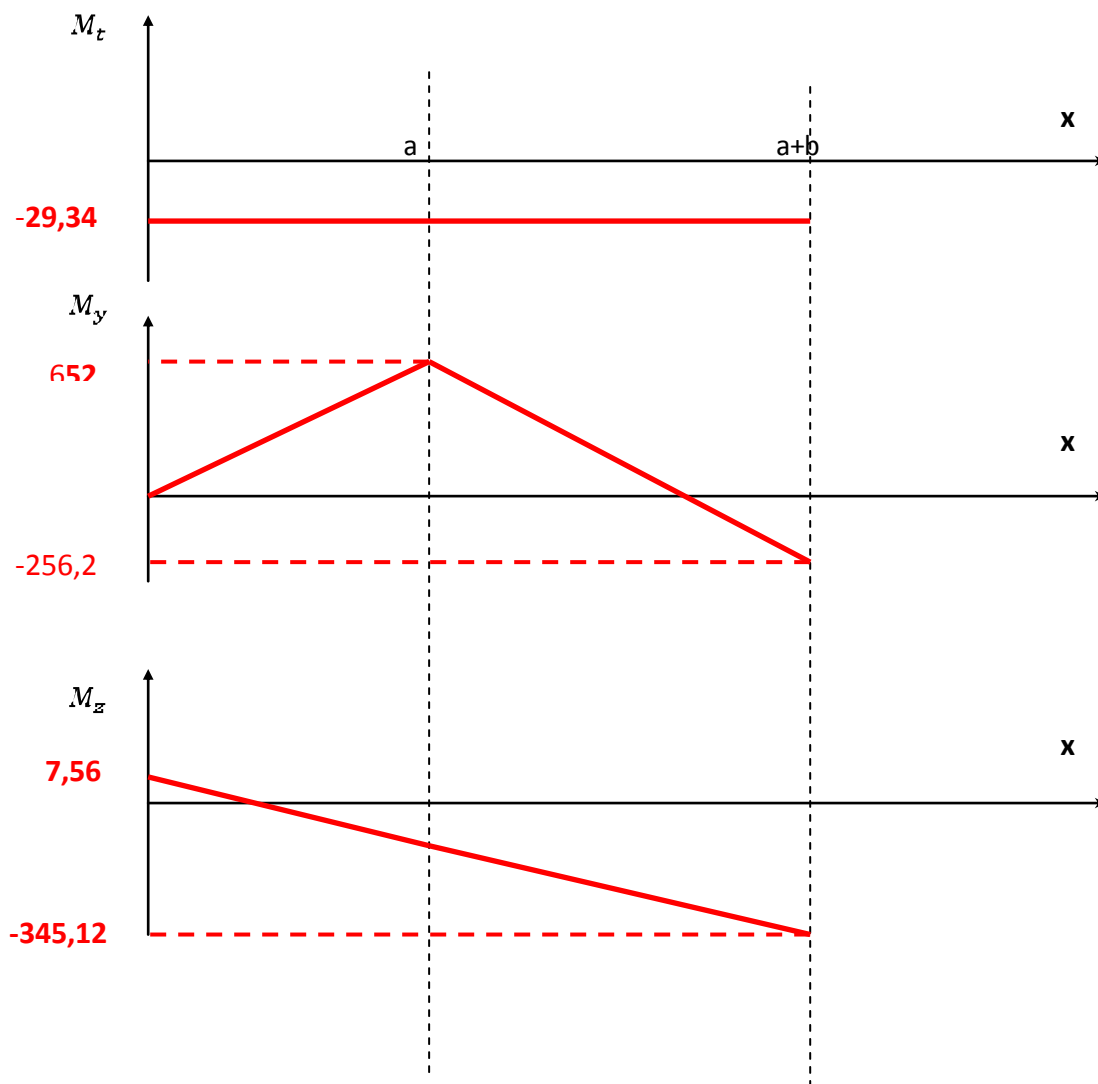
. Pour $0 \leq x \leq a$:

$$\vec{M}_I = \vec{R}_I \wedge \vec{IM} = \begin{cases} F_a \\ F_r \\ F_t \end{cases} \wedge \begin{cases} x \\ r \\ 0 \end{cases} = \begin{cases} -F_t \cdot r \\ F_t \cdot x \\ F_a \cdot r - F_r \cdot x \end{cases}$$

$$= \begin{cases} -29,34 \\ 652 \cdot x \\ 7,56 - 168 \cdot x \end{cases}$$

. Pour $a \leq x \leq b+a$:

$$\vec{M}_I(M) = \vec{R}_I \wedge \vec{IM} + \vec{RC} \wedge \vec{CM} = \begin{cases} M_x = -29,34 \\ M_y = -772x + 1365 \\ M_z = 7,68 - 168 \cdot x \end{cases}$$



On néglige les concentrations des contraintes

On va vérifier la résistance de l'arbre au sens de VAN MISES

$$\sigma_{eq} = \sqrt{\sigma_N^2 + 3\tau^2}$$

$$\sigma_{eq} = \left[\left(\frac{M_{fy}}{\frac{\pi D^4}{64}} \cdot \frac{D}{2} + \frac{M_{fz}}{\frac{\pi D^4}{64}} \cdot D/2 \right)^2 + 3 \cdot \left(\frac{M_t}{\frac{\pi D^4}{64}} \right)^2 \right]^{1/2} < \sigma_{ad} = Re$$